

PATENT SPECIFICATION

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COMPLETE SPECIFICATION

Rotor Assemblies for Gas Turbine Engines

SPECIFICATION NO. 813,522

By direction given under Section 17(1) of the Patents Act 1949 this application proceeded in the name of Canadian Patents and Development, Limited, of National Research Building, Sussex Street, City of Ottawa, County of Carleton, Province of Ontario, Canada, a Canadian company.

THE PATENT OFFICE,
2nd July, 1959

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ATIONS, ARE USED.

It is an object of the present invention to obviate these and other disadvantages and to provide a rotor assembly that is cheaply and easily assembled and has a high resistance to centrifugal stress.

It is a further object of the invention to provide a rotor assembly in which rotor blades may be mounted without the need for prior machining of the blade roots.

It is another object of the invention to provide in the rotor assembly means for cooling the rotor disc as well as the blades when the rotor assembly is rotated.

According to the invention, the rotor assembly comprises a rotor disc having a threaded cylindrical face, a blade ring having a threaded cylindrical face mating with the threaded face of the rotor disc, the blade ring having slots in its threaded face to receive a plurality of radially projecting rotor blades, the rotor disc when assembled overlying each of the slots in the blade ring

45 to close the slots against movement of the plurality of blade slots 14 extending through 90 [Price 3/6]

Fig. 1 is an exploded view of a rotor assembly for a gas turbine engine constructed according to the invention, also showing an assembling or dismantling tool, with a part 70 of the assembly cut off to show an assembling ring on the inside of the blade ring; and Fig. 2 is a perspective view of a segment of a slightly modified form of rotor assembly according to the invention showing a rotor 75 disc and a blade ring in assembled position, and a cranking tool for use in assembling and dismantling the rotor assembly.

Referring to the drawings, a rotor disc 10 has an externally threaded cylindrical flange 11 mated to the internally threaded flange 13 of a blade ring 12. The screw threads may either extend over the entire length of the circumferential surfaces of the flange or may be interrupted according to known threading practice. The hand of the thread is preferably opposite to the intended direction of rotation of the assembly or element.

The flange 13 of the blade ring has a

plurality of blade slots 14 extending through 90

RESE

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COMPLETE SPECIFICATION

Rotor Assemblies for Gas Turbine Engines

1. JOHN DAVID ALEXANDER MACKAY, of Orenda Engines Limited, Village of Malton, County of Peel, Province of Ontario, Canada, to the other.

According to a preferred embodiment of the invention, the rotor assembly comprises 50. a rotor disc having an externally threaded cylindrical flange; a blade ring having an internally threaded flange slotted to receive a plurality of rotor blades and adapted to mate with the threaded flange of the rotor disc, the 55 rotor disc overlying each of the slots in the blade ring to engage the blades and thus hold them in the slots.

The invention will now be described in more detail with reference to the accompanying drawings, it being understood that these are illustrative only and not intended to limit the scope of the invention.

In the drawings, in which each reference character denotes the same part in all the 65 views;

Fig. 1 is an exploded view of a rotor assembly for a gas turbine engine constructed according to the invention, also showing an assembling or dismantling tool, with a part 70 of the assembly cut off to show an assembling ring on the inside of the blade ring; and Fig. 2 is a perspective view of a segment of a slightly modified form of rotor assembly according to the invention showing a rotor 75 disc and a blade ring in assembled position, and a cranking tool for use in assembling and dismantling the rotor assembly.

It is a further object of the invention to provide a rotor assembly in which rotor 80 blades may be mounted without the need for prior machining of the blade roots.

It is another object of the invention to provide in the rotor assembly means for cooling the rotor disc as well as the blades 85 when the rotor assembly is rotated.

According to the invention, the rotor assembly comprises a rotor disc having a threaded cylindrical face, a blade ring having a threaded cylindrical face mating with the 90 threaded face of the rotor disc, the blade ring having slots in its threaded face to receive a plurality of rotor blades, the rotor disc when assembled 95 presenting a ring of teeth 23 which mesh with teeth 25 on a cranking tool having a shaft forming on one

its radial thickness, the slots being elongated in the axial direction of the rotor disc, as best shown in Fig. 2, and preferably they are tapered radially outwardly so that they have a lesser area on the outer than on the inner circumferential face of the flange.

Thus the slots are adapted to receive blades 15 preferably having tapered roots 15n. as shown in Fig. 2, that can be inserted from 10 into the flange of the blade ring and that are retained by the taper in the slots against removal by movement through the slots in an outward radial direction.

Instead of being tapered, the slots and 15 blade-roots may be otherwise shaped to provide other means for preventing removal of the blades by outward movement through the slots; for instance, the slots may be stepped inwardly near the outer face of the 20 ring, and the roots correspondingly shaped to engage the step in the slot.

The form of the invention shown in Fig. 2 has a circumferential groove 17 formed in the outer face of the flange 11 of the rotor disc, 25 and each of the blades 15 has an open passageway extending between its outer end 30 and the inner face of the blade root. A circumferential row of the blades is radially disposed in the slots 14. When the blades are in position, the passageways in the blades are in communication with the groove 17 in the rotor disc. Passages 18 extending through the side face of the blade ring 12 lead into the groove to receive cooling air 35 which may be supplied by known structures to the space adjacent the rotor disc. The air entering the groove passes out through the passageways in the blades.

For convenience, removable means are 40 provided to hold the blades temporarily in the blade ring while the ring is assembled on the rotor disc; such means may comprise an assembling ring 19 as shown in Fig. 1, preferably of spring wire, adapted to be 45 reduced in diameter and passed under the blade roots and allowed to expand against the inner circumference of the blade ring to engage the roots and maintain them in position while the ring is secured to or removed from the rotor disc.

Also for convenience, means are provided for applying a force to the blade ring to screw it onto the rotor disc. As shown in Fig. 1, a special tool 20 provides lug 20a arranged so as to enter slots 21 in the side face of the blade ring; a handle 21b on the tool provides sufficient leverage for tightening the blade ring on the rotor disc.

A more convenient means for tightening the elements of the assembly, together is shown in Fig. 2, the rim of the blade ring 12 is provided with an inner circumferential shoulder presenting a ring of teeth 23 which mesh with teeth 25 on a cranking tool 26. The cranking tool has a shaft forming on one



sake a handle 27 and on the other side an axle that may be inserted in a bearing hole in the face of the rotor disc. Rotation of the cranking tool causes rotation of the blade ring on the rotor disc until it is tightened or loosened.

When assembling the rotor assembly or element by threading the flanges together, advantage may be taken of the working temperature difference between them, by 75 arranging that the internally threaded flange of the cooler part (the ring) overlaps on the outside the externally threaded flange of the hotter part, (the rotor disc) thereby ensuring that any differential expansion will tend to 80 keep the threaded joint tight.

The foregoing description sets forth the best mode contemplated by the inventor of carrying out his invention, but the following Claims are intended to cover all useful changes, 85 and modifications of the said mode which are within the scope of the invention.

WHAT I CLAIM IS:—

1. A rotor assembly for a gas turbine

engine comprising a rotor disc having a 90

threaded cylindrical face, a blade ring having a 95

threaded cylindrical face mating with the 100 threaded face of the rotor disc, the blade ring having slots in its threaded face to receive a plurality of rotor blades and means for rotating one of the threaded faces relatively 105 to the other.

2. A rotor assembly as claimed in Claim 1

in which the threaded face of the rotor disc 110

is on an externally threaded cylindrical 115

flange and the blade ring has an internally 120

threaded flange mating with the flange of the 125

rotor disc.

3. A rotor assembly as claimed in Claim 1

or 2 including means for preventing move- 130

ment of the blades away from the rotor disc.

4. A rotor assembly as claimed in Claim 3

in which the means for preventing move- 135

ment of the blades away from the rotor disc 140

comprises tapered walls of the slots and 145

correspondingly tapered roots on the blades 150

which fit; blades can be removed from the 155

blade ring by radially inward movement.

5. A rotor assembly as claimed in any of 160

the preceding Claims in which the means for 165

rotating one of the threaded faces relatively 170

to the other comprises a tool having long 175

teeth 25 on a cranking tool 26.

6. The cranking tool has a shaft forming on one 180

end of the shaft 185

and on the other side an 190

axle that may be inserted in a bearing hole 195

in the face of the rotor disc.

Rotation of the cranking tool causes 200

rotation of the blade ring on the rotor disc.

7. A rotor assembly as claimed in any of 205

the preceding Claims in which the means for 210

rotating one of the threaded faces relatively 215

to the other comprises a tool having long 220

teeth 25 on a cranking tool 26.

8. The cranking tool has a shaft forming on one 225

end of the shaft 230

and on the other side an 235

axle that may be inserted in a bearing hole 235

in the face of the rotor disc.

Rotation of the cranking tool causes 240

rotation of the blade ring on the rotor disc.

9. A rotor assembly as claimed in any of 245

the preceding Claims in which the means for 250

rotating one of the threaded faces relatively 255

to the other comprises a tool having long 260

teeth 25 on a cranking tool 26.

10. The cranking tool has a shaft forming on one 265

end of the shaft 270

and on the other side an 275

axle that may be inserted in a bearing hole 280

in the face of the rotor disc.

Rotation of the cranking tool causes 285

rotation of the blade ring on the rotor disc.

11. A rotor assembly as claimed in any of 290

the preceding Claims in which the means for 295

rotating one of the threaded faces relatively 300

to the other comprises a tool having long 305

teeth 25 on a cranking tool 26.

12. The cranking tool has a shaft forming on one 310

end of the shaft 315

and on the other side an 320

axle that may be inserted in a bearing hole 325

in the face of the rotor disc.

Rotation of the cranking tool causes 330

rotation of the blade ring on the rotor disc.

13. A rotor assembly as claimed in any of 335

the preceding Claims in which the means for 340

rotating one of the threaded faces relatively 345

to the other comprises a tool having long 350

teeth 25 on a cranking tool 26.

14. The cranking tool has a shaft forming on one 355

end of the shaft 360

and on the other side an 365

axle that may be inserted in a bearing hole 370

in the face of the rotor disc.

Rotation of the cranking tool causes 375

rotation of the blade ring on the rotor disc.

15. A rotor assembly as claimed in any of 380

the preceding Claims in which the means for 385

rotating one of the threaded faces relatively 390

to the other comprises a tool having long 395

teeth 25 on a cranking tool 26.

16. The cranking tool has a shaft forming on one 400

end of the shaft 405

and on the other side an 410

axle that may be inserted in a bearing hole 415

in the face of the rotor disc.

Rotation of the cranking tool causes 420

rotation of the blade ring on the rotor disc.

17. A rotor assembly as claimed in any of 425

the preceding Claims in which the means for 430

rotating one of the threaded faces relatively 435

to the other comprises a tool having long 440

teeth 25 on a cranking tool 26.

18. The cranking tool has a shaft forming on one 445

end of the shaft 450

and on the other side an 455

axle that may be inserted in a bearing hole 460

in the face of the rotor disc.

Rotation of the cranking tool causes 465

rotation of the blade ring on the rotor disc.

19. A rotor assembly as claimed in any of 470

the preceding Claims in which the means for 475

rotating one of the threaded faces relatively 480

to the other comprises a tool having long 485

teeth 25 on a cranking tool 26.

20. The cranking tool has a shaft forming on one 490

end of the shaft 495

and on the other side an 500

axle that may be inserted in a bearing hole 505

in the face of the rotor disc.

Rotation of the cranking tool causes 510

rotation of the blade ring on the rotor disc.

21. A rotor assembly as claimed in any of 515

the preceding Claims in which the means for 520

rotating one of the threaded faces relatively 525

to the other comprises a tool having long 530

teeth 25 on a cranking tool 26.

22. The cranking tool has a shaft forming on one 535

end of the shaft 540

and on the other side an 545

axle that may be inserted in a bearing hole 550

in the face of the rotor disc.

Rotation of the cranking tool causes 555

rotation of the blade ring on the rotor disc.

23. A rotor assembly as claimed in any of 560

the preceding Claims in which the means for 565

rotating one of the threaded faces relatively 570

to the other comprises a tool having long 575

teeth 25 on a cranking tool 26.

24. The cranking tool has a shaft forming on one 580

end of the shaft 585

and on the other side an 590

axle that may be inserted in a bearing hole 595

in the face of the rotor disc.

Rotation of the cranking tool causes 600

rotation of the blade ring on the rotor disc.

25. A rotor assembly as claimed in any of 605

the preceding Claims in which the means for 610

rotating one of the threaded faces relatively 615

to the other comprises a tool having long 620

teeth 25 on a cranking tool 26.

26. The cranking tool has a shaft forming on one 625

end of the shaft 630

and on the other side an 635

axle that may be inserted in a bearing hole 640

in the face of the rotor disc.

Rotation of the cranking tool causes 645

rotation of the blade ring on the rotor disc.

27. A rotor assembly as claimed in any of 650

the preceding Claims in which the means for 655

rotating one of the threaded faces relatively 660

to the other comprises a tool having long 665

teeth 25 on a cranking tool 26.

28. The cranking tool has a shaft forming on one 670

end of the shaft 675

and on the other side an 680

axle that may be inserted in a bearing hole 685

in the face of the rotor disc.

Rotation of the cranking tool causes 690

rotation of the blade ring on the rotor disc.

29. A rotor assembly as claimed in any of 695

the preceding Claims in which the means for 700

rotating one of the threaded faces relatively 705

to the other comprises a tool having long 710

teeth 25 on a cranking tool 26.

30. The cranking tool has a shaft forming on one 715

end of the shaft 720

and on the other side an 725

axle that may be inserted in a bearing hole 730

in the face of the rotor disc.

Rotation of the cranking tool causes 735

rotation of the blade ring on the rotor disc.

31. A rotor assembly as claimed in any of 740

the preceding Claims in which the means for 745

rotating one of the threaded faces relatively 750

to the other comprises a tool having long 755

teeth 25 on a cranking tool 26.

32. The cranking tool has a shaft forming on one 760

end of the shaft 765

and on the other side an 770

axle that may be inserted in a bearing hole 775

in the face of the rotor disc.

Rotation of the cranking tool causes 780

rotation of the blade ring on the rotor disc.

33. A rotor assembly as claimed in any of 785

the preceding Claims in which the means for 790

rotating one of the threaded faces relatively 795

to the other comprises a tool having long 800

providing a handle and an axle, the rotor disc having a bearing hole for the axle of the tool disposed so that the teeth on the tool and the ring are in operative engagement whereby rotation of the handle rotates the blade ring.

7. A rotor assembly as claimed in any of the preceding Claims in which the threaded face of the rotor disc or of the blade ring has a circumferential groove communicating when 10 the blade ring is threaded on the rotor disc with the exterior of the assembly, and in which each of the rotor blades has a passage extending from an opening at the tip of the blade to an opening at the radially inner end 15 of the blade communicating with the groove whereby cooling air entering the groove will pass out through the passageways in the blades.

8. A rotor assembly as claimed in any of the preceding Claims including an expandable elastic annular element having a diameter normally greater than the inner

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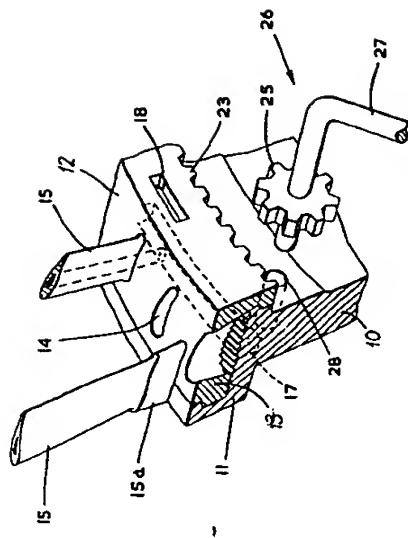


FIG. 2

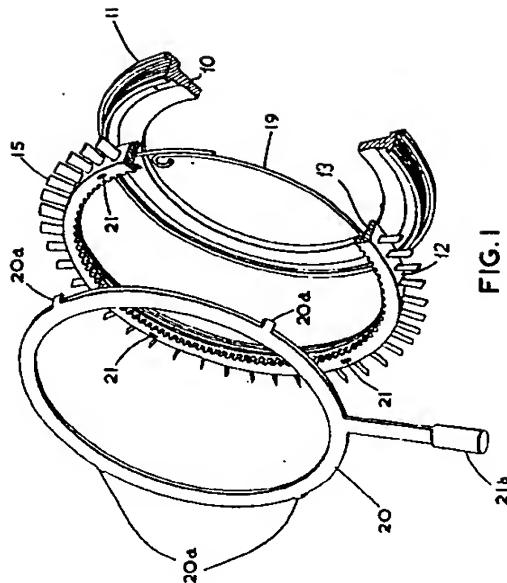


FIG. 1